

Appln. No. 10/743,394  
Amdt. Dated Mar. 17, 2005  
Reply to OA of Sep. 17, 2004

**Amendments to the Specification:**

Please replace paragraphs [020] and [021] with the following rewritten paragraphs:

[020] Figure 5 is a plan view of the evaporator of Figure 3 taken along line 4-4 of Figure 3; and

[021] Figure 6 is a perspective view of an alternate embodiment of the evaporator of the present invention; and

Please add the following new paragraph after the paragraph [021]:

[021.1] Figure 7 is a plan view of an alternative embodiment of the evaporator of the present invention.

Please replace paragraph [033] with the following rewritten paragraph:

[033] As the heat transfer surface of the second heat exchanger coil 40B is advantageously maintained at a temperature above the freezing point of water, frost formation will not be a problem in the second heat exchanger coil 30B. Accordingly, the second heat exchanger coil 40B may have a relatively high fin density, that is a fin density of at least 6 fins per inch, to improve and /or optimize heat transfer between the refrigerant and the circulating air. As frost is likely to occur on the colder heat transfer surfaces of the first heat exchanger coil 40A, the first heat exchanger coil will have a relatively low fin density, that is a fin density of less than 6 fins per inch. The first heat exchanger coil 40A may even be a non-finned, bare tube coil, which would have a fin density of zero, as illustrated in Figure 7. Having a low fin density, frost may accumulate to a greater extent without significant degradation in evaporator performance.